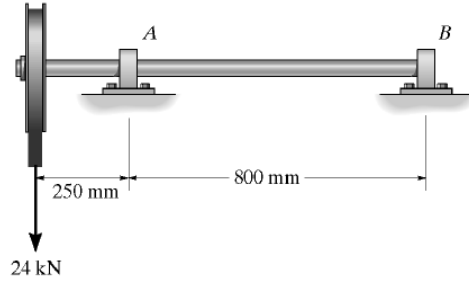
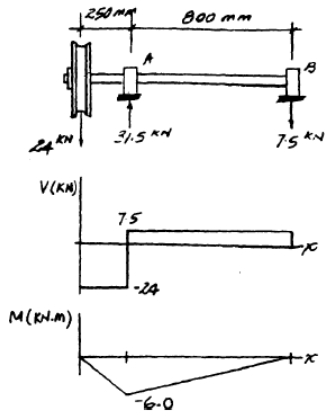
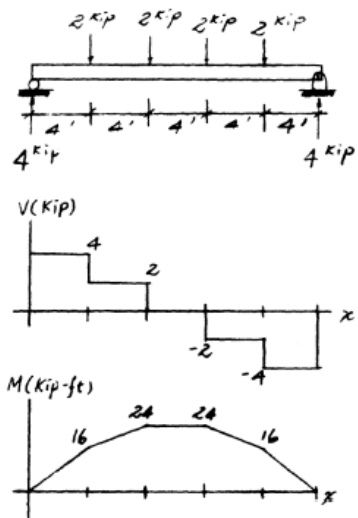
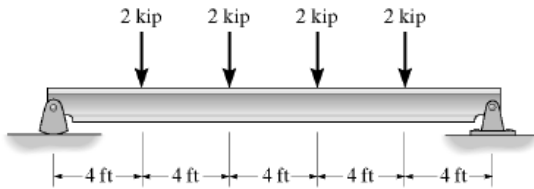


HW 19 SOLUTIONS

6-1. Draw the shear and moment diagrams for the shaft. The bearings at *A* and *B* exert only vertical reactions on the shaft.



*6-4. Draw the shear and moment diagrams for the beam.

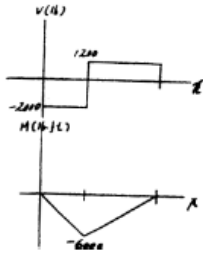
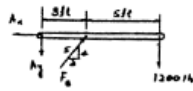
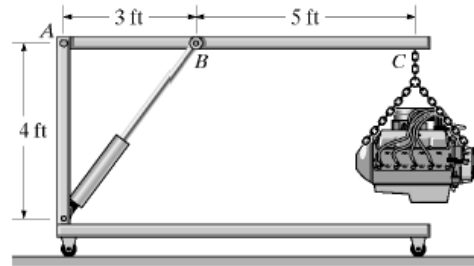


6-10. The engine crane is used to support the engine, which has a weight of 1200 lb. Draw the shear and moment diagrams of the boom ABC when it is in the horizontal position shown.

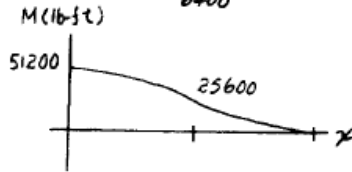
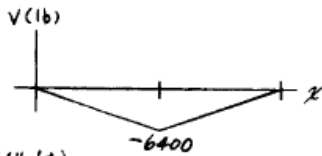
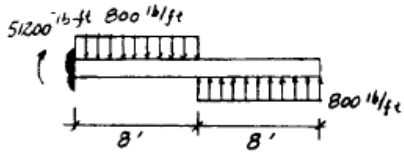
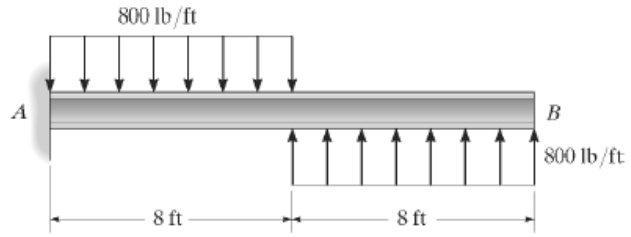
$$\zeta + \Sigma M_A = 0; \quad \frac{4}{5}F_B(3) - 1200(8) = 0; \quad F_B = 4000 \text{ lb}$$

$$+ \uparrow \Sigma F_y = 0; \quad -A_y + \frac{4}{5}(4000) - 1200 = 0; \quad A_y = 2000 \text{ lb}$$

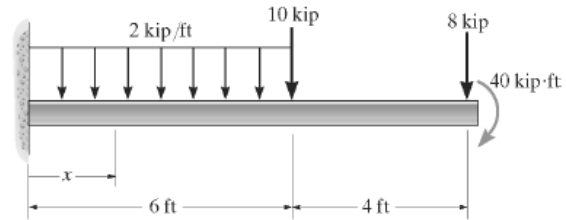
$$\rightarrow \Sigma F_x = 0; \quad A_x - \frac{3}{5}(4000) = 0; \quad A_x = 2400 \text{ lb}$$



*6-16. Draw the shear and moment diagrams for the beam.



*6-20. Draw the shear and moment diagrams for the beam, and determine the shear and moment throughout the beam as functions of x .

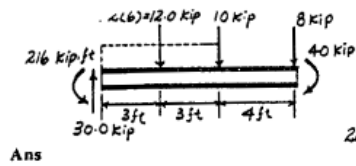


Support Reactions: As shown on FBD.
Shear and Moment Function:

For $0 \leq x < 6$ ft:

$$+\uparrow \Sigma F_y = 0; \quad 30.0 - 2x - V = 0$$

$$V = (30.0 - 2x) \text{ kip}$$



$$\curvearrowleft + \Sigma M_{NA} = 0; \quad M + 216 + 2x\left(\frac{x}{2}\right) - 30.0x = 0$$

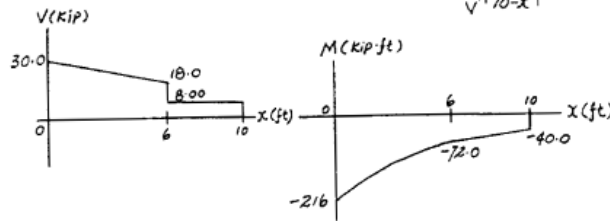
$$M = (-x^2 + 30.0x - 216) \text{ kip} \cdot \text{ft}$$

For $6 \text{ ft} < x \leq 10 \text{ ft}$:

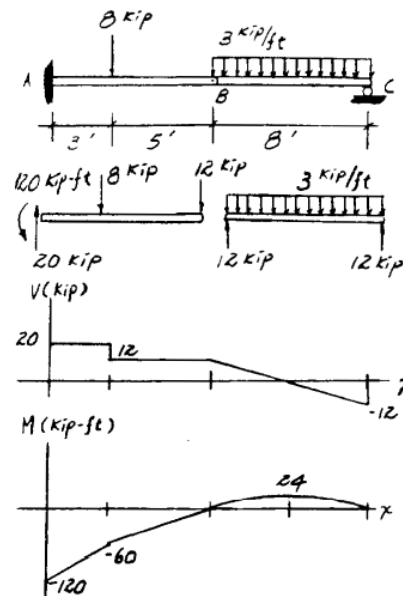
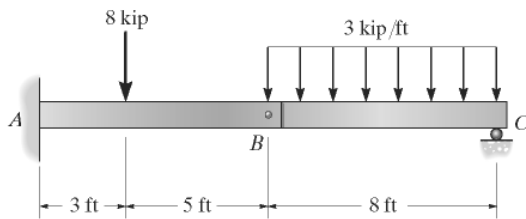
$$+\uparrow \Sigma F_y = 0; \quad V - 8 = 0 \quad V = 8.00 \text{ kip}$$

$$\curvearrowleft + \Sigma M_{NA} = 0; \quad -M - 8(10 - x) - 40 = 0$$

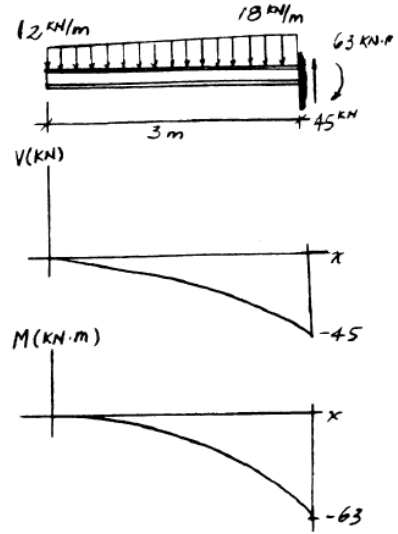
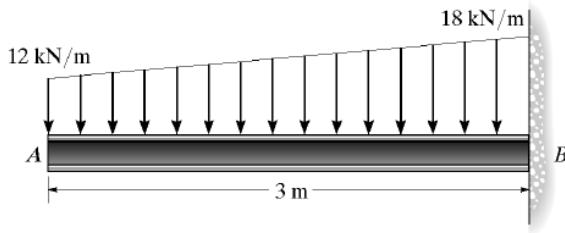
$$M = (8.00x - 120) \text{ kip} \cdot \text{ft}$$



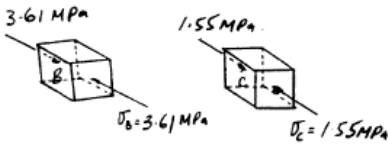
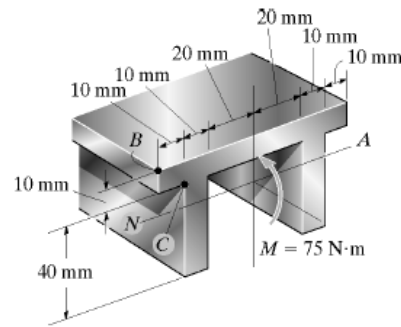
6-25. Draw the shear and moment diagrams for the beam. The two segments are joined together at B.



6-38. Draw the shear and moment diagrams for the beam.



6-51. The aluminum machine part is subjected to a moment of $M = 75 \text{ N}\cdot\text{m}$. Determine the bending stress created at points B and C on the cross section. Sketch the results on a volume element located at each of these points.



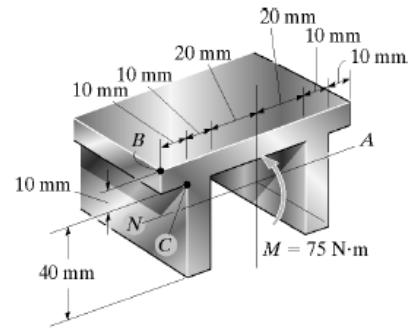
$$\bar{y} = \frac{0.005(0.08)(0.01) + 2[0.03(0.04)(0.01)]}{0.08(0.01) + 2(0.04)(0.01)} = 0.0175 \text{ m}$$

$$I = \frac{1}{12}(0.08)(0.01^3) + 0.08(0.01)(0.0125^2) + 2[\frac{1}{12}(0.01)(0.04^3) + 0.01(0.04)(0.0125^2)] = 0.3633(10^{-6}) \text{ m}^4$$

$$\sigma_B = \frac{Mc}{I} = \frac{75(0.0175)}{0.3633(10^{-6})} = 3.61 \text{ MPa} \quad \text{Ans}$$

$$\sigma_C = \frac{My}{I} = \frac{75(0.0175 - 0.01)}{0.3633(10^{-6})} = 1.55 \text{ MPa} \quad \text{Ans}$$

*6-52. The aluminum machine part is subjected to a moment of $M = 75 \text{ N} \cdot \text{m}$. Determine the maximum tensile and compressive bending stresses in the part.



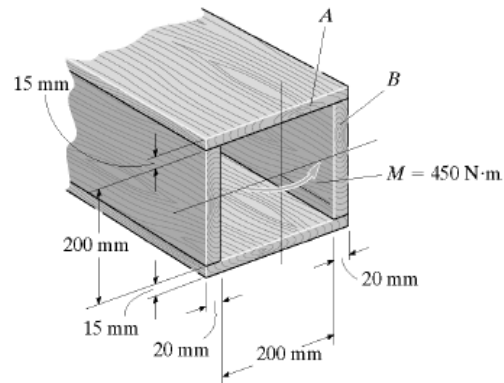
$$\bar{y} = \frac{0.005(0.08)(0.01) + 2[0.03(0.04)(0.01)]}{0.08(0.01) + 2(0.04)(0.01)} = 0.0175 \text{ m}$$

$$I = \frac{1}{12}(0.08)(0.01^3) + 0.08(0.01)(0.0125^2) + 2\left[\frac{1}{12}(0.01)(0.04^3) + 0.01(0.04)(0.0125^2)\right] = 0.3633(10^{-6}) \text{ m}^4$$

$$(\sigma_{\max})_t = \frac{Mc}{I} = \frac{75(0.050 - 0.0175)}{0.3633(10^{-6})} = 6.71 \text{ MPa} \quad \text{Ans}$$

$$(\sigma_{\max})_c = \frac{My}{I} = \frac{75(0.0175)}{0.3633(10^{-6})} = 3.61 \text{ MPa} \quad \text{Ans}$$

6-53. A beam is constructed from four pieces of wood, glued together as shown. If the moment acting on the cross section is $M = 450 \text{ N} \cdot \text{m}$, determine the resultant force the bending stress produces on the top board A and on the side board B.



$$I_y = \frac{1}{12}(0.23)(0.24^3) - \frac{1}{12}(0.2)(0.2^3) = 1.31626(10^{-4}) \text{ m}^4$$

$$\sigma_D = \frac{Mx}{I_y} = \frac{450(0.12)}{1.31626(10^{-4})} = 410.25 \text{ kPa}$$

$$\sigma_C = \frac{Mx}{I_y} = \frac{450(0.1)}{1.31626(10^{-4})} = 341.88 \text{ kPa}$$



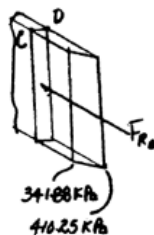
$$F_{R_1} = F_{R_2} - F_{R_2}$$

$$= \frac{1}{2}(410.25)(10^3)(0.12)(0.015) - \frac{1}{2}(410.25)(10^3)(0.12)(0.015)$$

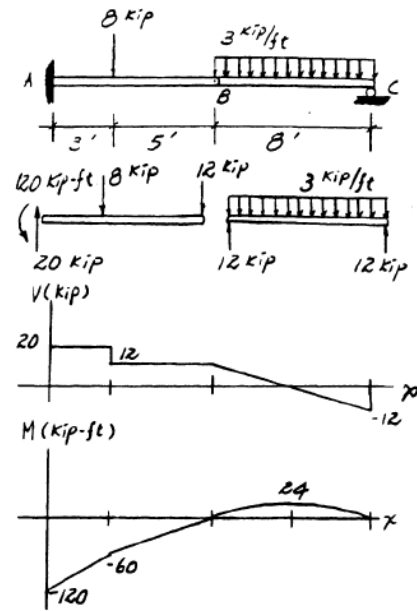
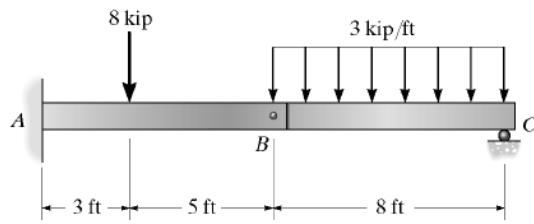
$$= 0 \quad \text{Ans}$$

$$F_{R_2} = 341.88(10^3)(0.2)(0.02) + \frac{1}{2}(410.25 - 341.88)(10^3)(0.2)(0.02)$$

$$= 1.50 \text{ kN} \quad \text{Ans}$$



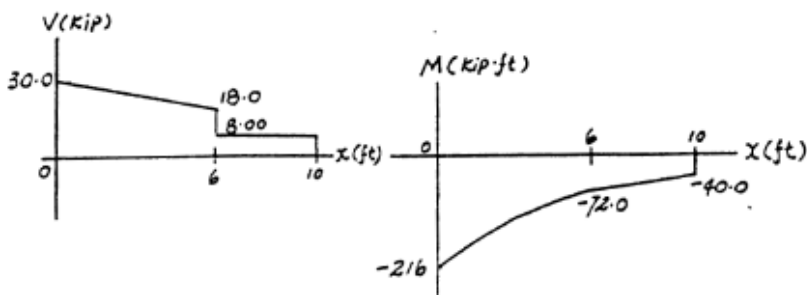
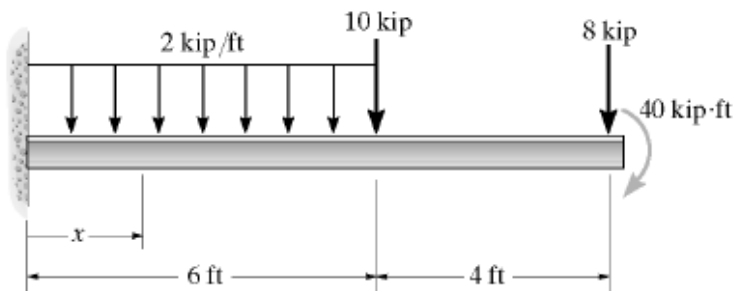
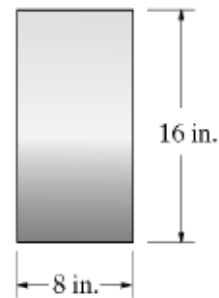
6-69. Determine the absolute maximum bending stress in the beam in Prob. 6-25. Each segment has a rectangular cross section with a base of 4 in. and height of 8 in.



$$M_{\max} = 120 \text{ kip} \cdot \text{ft}$$

$$\sigma_{\max} = \frac{Mc}{I} = \frac{120(12)(10^3)(4)}{\frac{1}{12}(4)(8)^3} = 33.8 \text{ ksi} \quad \text{Ans}$$

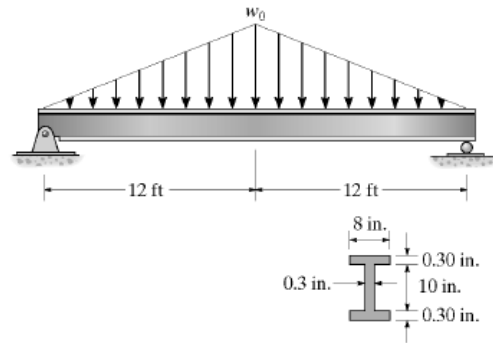
6-78. If the beam in Prob. 6-20 has a rectangular cross section with a width of 8 in. and a height of 16 in., determine the absolute maximum bending stress in the beam.



Absolute Maximum Bending Stress : The maximum moment is $M_{\max} = 216$ kip · ft as indicated on moment diagram. Applying the flexure formula

$$\sigma_{\max} = \frac{M_{\max} c}{I} = \frac{216(12)(8)}{\frac{1}{12}(8)(16^3)} = 7.59 \text{ ksi} \quad \text{Ans}$$

6-89. The steel beam has the cross-sectional area shown. If $w_0 = 0.5$ kip/ft, determine the maximum bending stress in the beam.



Support Reactions: As shown on FBD.

Internal Moment: The maximum moment occurs at mid span. The maximum moment is determined using the method of sections.

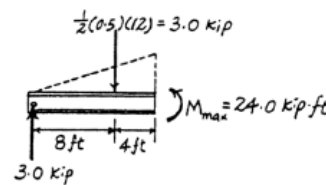
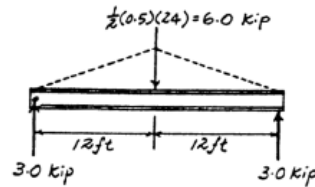
Section Property:

$$I = \frac{1}{12}(8)(10.6^3) - \frac{1}{12}(7.7)(10^3) = 152.344 \text{ in}^4$$

Absolute Maximum Bending Stress: The maximum moment is $M_{\max} = 24.0$ kip · ft as indicated on the FBD. Applying the flexure formula

$$\begin{aligned} \sigma_{\max} &= \frac{M_{\max} c}{I} \\ &= \frac{24.0(12)(5.30)}{152.344} \\ &= 10.0 \text{ ksi} \end{aligned}$$

Ans



6-146. The bar is subjected to a moment of $M = 17.5 \text{ N} \cdot \text{m}$. If $r = 5 \text{ mm}$, determine the maximum bending stress in the material.

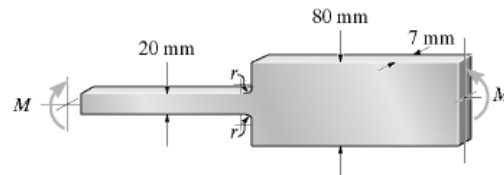
Stress Concentration Factor: From the graph in the text

with $\frac{w}{h} = \frac{80}{20} = 4$ and $\frac{r}{h} = \frac{5}{20} = 0.25$, then $K = 1.45$.

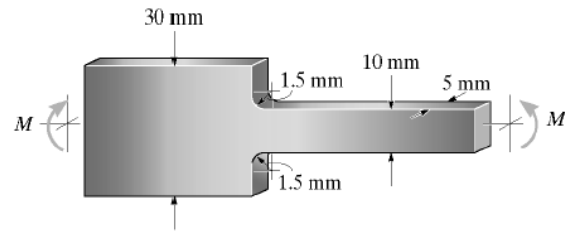
Maximum Bending Stress:

$$\begin{aligned} \sigma_{\max} &= K \frac{Mc}{I} \\ &= 1.45 \left[\frac{17.5(0.01)}{\frac{1}{12}(0.007)(0.02^3)} \right] \\ &= 54.4 \text{ MPa} \end{aligned}$$

Ans



*6-148. The allowable bending stress for the bar is $\sigma_{\text{allow}} = 175 \text{ MPa}$. Determine the maximum moment M that can be applied to the bar.



Stress Concentration Factor: From the graph in the text with $\frac{w}{h} = \frac{30}{10} = 3$ and $\frac{r}{h} = \frac{1.5}{10} = 0.15$, then $K = 1.6$.

Maximum Bending Stress:

$$\sigma_{\text{max}} = \sigma_{\text{allow}} = K \frac{Mc}{I}$$

$$175 (10^6) = 1.6 \left[\frac{M(0.005)}{\frac{1}{12}(0.005)(0.01^3)} \right]$$

$$M = 9.11 \text{ N} \cdot \text{m} \quad \text{Ans}$$