

2.1 (a) $\delta = \frac{PL}{AE} = \frac{(800 \text{ lb})(12 \text{ in.})}{\pi (0.25 \text{ in.})^2 (0.45 \times 10^6 \text{ psi})}$
 $\delta = 0.1086 \text{ in.}$

(b) $G = \frac{P}{A} = \frac{800 \text{ lb}}{\pi (0.25 \text{ in.})^2}$
 $G = 4.07 \text{ ksi}$

2.67 $\epsilon_y = \frac{\delta}{L} = \frac{14 \text{ mm}}{150 \text{ mm}} = 0.09333$

$\epsilon_z = \frac{\Delta d}{d} = \frac{-0.85 \text{ mm}}{20 \text{ mm}} = -0.0425$

$\nu = -\frac{\epsilon_z}{\epsilon_y} = -\frac{-0.0425}{0.09333} = 0.45536, \nu = 0.455$

$\sigma_y = \frac{P}{A} = \frac{6 \times 10^3 \text{ N}}{\pi (0.01 \text{ m})^2} = 19.099 \text{ MPa}$

$E = \frac{\sigma_y}{\epsilon_y} = \frac{19.099 \text{ MPa}}{0.09333} = 204.63 \text{ MPa}$ $E = 205 \text{ MPa}$

From Eq. (2.43'): $G = \frac{E}{2(1+\nu)}$

$G = \frac{204.63 \text{ MPa}}{2(1+0.45536)}$ $G = 70.3 \text{ MPa}$

3.2 $\tau = \frac{Tc}{J}$: $T = \frac{\tau J}{c}$

(a) $T = \frac{\tau J}{c} = \frac{45 \times 10^6 \text{ Pa} \cdot \pi [(0.045 \text{ m})^4 - (0.030 \text{ m})^4]}{0.045 \text{ m}} = 5.1689 \times 10^3$
 $T = 5.17 \text{ kN}\cdot\text{m}$

(b) $J = \pi [(45 \text{ mm})^4 - (30 \text{ mm})^4] = \pi c^2$ $c = 33.541 \text{ mm}$

$\tau = \frac{Tc}{J} = \frac{Tc}{\pi c^2} = \frac{2}{\pi} \frac{T}{c^3} = \frac{2}{\pi} \frac{5.1689 \times 10^3 \text{ N}\cdot\text{m}}{(33.541 \times 10^{-3} \text{ m})^3}$
 $\tau = 87.2 \text{ MPa}$

3.15 Shaft AB: $\tau = 50 \text{ MPa}$ $c = 0.020 \text{ m}$

Using Eq. (1) of Prob. 3.9:
 $T_{AB} = \frac{\pi}{2} c^3 \tau = \frac{\pi}{2} (0.020 \text{ m})^3 (50 \times 10^6 \text{ Pa}) = 628 \text{ N}\cdot\text{m}$

Shaft CD: $\tau = 50 \text{ MPa}$ $c = 0.030 \text{ m}$

$T_{CD} = \frac{\pi}{2} c^3 \tau = \frac{\pi}{2} (0.030 \text{ m})^3 (50 \times 10^6 \text{ Pa}) = 2121 \text{ N}\cdot\text{m}$

Recalling Eq. (3) of solution of Prob. 3.13:
 $T_{AB} = \frac{\tau_{AB}}{\tau_{CD}} T_{CD} = \frac{100 \text{ mm}}{250 \text{ mm}} (2121 \text{ N}\cdot\text{m}) = 848 \text{ N}\cdot\text{m}$

Stress in shaft AB is critical: $T = 628 \text{ N}\cdot\text{m}$

2.5-8 $\delta_B = -\delta_S$ $P_B = -P_S = \sigma_B A_B = \sigma_S A_S$

$\frac{P_B K}{EA_B} + \alpha_B \Delta T L = \frac{P_S K}{EA_S} + \alpha_S \Delta T L$ Because $L_B = L_S = L$

$\frac{\sigma_B A_B K}{EA_B} + \alpha_B \Delta T = \frac{\sigma_S A_S K}{EA_S} + \alpha_S \Delta T$

But $\sigma_B = -\sigma_S A_S / A_B \rightarrow \frac{-\sigma_S A_S K}{EA_B} + \alpha_B \Delta T = \frac{\sigma_S A_S K}{EA_S} + \alpha_S \Delta T$

$-\sigma_S \left[\frac{A_S}{A_B} \frac{1}{E_B} + \frac{1}{E_S} \right] = (\alpha_S - \alpha_B) \Delta T$ Bad $A_S = \frac{\pi}{4} (0.030^2 - 0.020^2)$

$A_S = 4.869 \times 10^{-4} \text{ m}^2$ $A_B = \frac{\pi}{4} (0.025)^2 = 4.908 \times 10^{-4} \text{ m}^2$

$A_S / A_B = 0.9919$ $\Delta T = \sigma_S \left[\frac{A_S}{A_B} \frac{1}{E_B} + \frac{1}{E_S} \right] \frac{1}{\alpha_B - \alpha_S}$

$\Delta T = -75 \times 10^6 \left[\frac{0.9919}{200 \times 10^9} + \frac{1}{100 \times 10^9} \right] \frac{1}{10 \times 10^{-6} - 21 \times 10^{-6}}$

$\Delta T = 33.998 = 34^\circ \text{C}$

Note subscript S for sleeve B to P shaft

2.13 $d_{AB} = d_{BC} = 2.25 \text{ in}$
 $d_{CD} = 1.75 \text{ in}$ $E = 10 \times 10^6 \text{ psi}$

Portion	P_i	L_i	A_i	$\frac{P_i L_i}{A_i E}$
CD	10×10^3	15	2.405	6.236×10^{-3}
BC	-20×10^3	8	3.976	-4.024×10^{-3}
AB	0	12	3.976	0

(a) $\delta_{AB} = \sum \frac{P_i L_i}{A_i E} = 6.236 \times 10^{-3} - 4.024 \times 10^{-3} = +2.21 \times 10^{-3} \text{ in.}$

(b) $\delta_C = -4.024 \times 10^{-3} + 0$ $\delta_C = 4.024 \times 10^{-3} \text{ in.}$

2.98 (a) Considering the hole: $t = \frac{3}{4} \text{ in.}$
 $D = b = 3 \text{ in.}$ $d = 3 - 0.6 = 2.4 \text{ in.}$ $t = \frac{1}{2} (0.6) = 0.3 \text{ in.}$
 $\frac{t}{d} = \frac{0.3 \text{ in.}}{2.4 \text{ in.}} = 0.125$ Fig. 2.59 $\rightarrow K = 2.45$

$\sigma_{max} = K \frac{P}{A}$: $P = \frac{A \sigma_{max}}{K} = \frac{t d \sigma_{all}}{K}$ with $\sigma_{all} = 24 \text{ ksi}$

$P = \frac{(\frac{3}{4} \text{ in.})(2.4 \text{ in.})(24 \text{ ksi})}{2.45}$ $P = 17.63 \text{ kips}$

(b) Considering the fillet: $d = b = 3 \text{ in.}$ $t = \frac{3}{4} \text{ in.}$

$\sigma_{max} = K \frac{P}{A}$: $K = \frac{A \sigma_{max}}{P} = \frac{t d \sigma_{all}}{P}$ with $\sigma_{all} = 24 \text{ ksi}$

$K = \frac{(\frac{3}{4} \text{ in.})(3 \text{ in.})(24 \text{ ksi})}{17.63 \text{ kips}} = 3.06$

From Fig. 2.59 for $K = 3.06$ and $\frac{D}{d} = \frac{4.5 \text{ in.}}{3 \text{ in.}} = 1.5$,
we obtain $t_f/d = 0.05$
Thus: $t_f = 0.05 d = 0.05 (3 \text{ in.})$ $t_f = 0.15 \text{ in.}$

3.23 (a) Hollow shaft

$J = \frac{\pi}{2} (c_2^4 - c_1^4) = \frac{\pi}{2} [(0.045 \text{ m})^4 - (0.030 \text{ m})^4] = 5.1689 \times 10^{-6}$

$\phi = \frac{TL}{JG}$ $T = \frac{JG \phi}{L} = \frac{(5.1689 \times 10^{-6}) (77 \times 10^9) (3^\circ \times \frac{\pi \text{ rad}}{180^\circ})}{2.4}$
 $T = 8.6831 \times 10^3 \text{ N}\cdot\text{m}$ $T = 8.68 \text{ kN}\cdot\text{m}$

(b) Solid shaft of same cross-sectional area

$A = \pi c^2 = \pi (c_2^2 - c_1^2)$ $c^2 = c_2^2 - c_1^2 = (45 \text{ mm})^2 - (30 \text{ mm})^2$
 $c^2 = 1125 \text{ mm}^2$

For solid shaft: $\phi_s = \frac{TL}{J_s G}$ (same T, same L, same G)

For hollow shaft: $\phi_h = \frac{TL}{J_h G}$

$\frac{\phi_s}{\phi_h} = \frac{J_h}{J_s} = \frac{\frac{\pi}{2} (c_2^4 - c_1^4)}{\frac{\pi}{2} c^4} = \frac{c_2^4 - c_1^4}{(c^2)^2} = \frac{(45)^4 - (30)^4}{(1125)^2} = 2.600$

Thus: $\phi_s = 2.6 \phi_h = 2.6 (3^\circ)$ $\phi_s = 7.80^\circ$

Alternative computation:
 $\phi_s = \frac{TL}{J_s G} = \frac{(8.6831 \times 10^3) (2.4)}{\frac{\pi}{2} (1125 \times 10^{-6})^2 (77 \times 10^9)} = 0.13614 \text{ rad}$
 $\phi_s = (0.13614 \text{ rad}) \frac{180^\circ}{\pi} = 7.80^\circ$