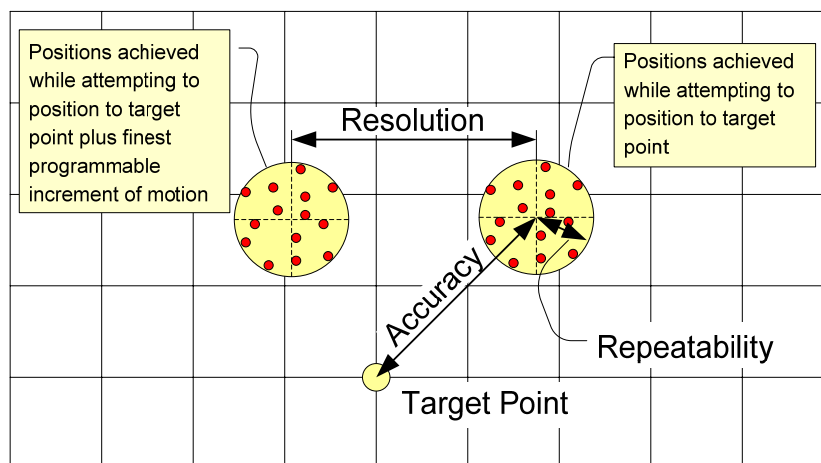


Fundamentals of Errors

ME EN 7960 – Precision Machine Design
Topic 2



Accuracy, Repeatability, and Resolution



Accuracy

- Accuracy is the ability to tell the truth or:
 - The maximum translational or rotational error between any two points in the machine's work volume
 - Linear, planar, and volumetric accuracy can all be defined for a machine
- Note: mechanical accuracy is far more costly than repeatability



Repeatability

- Repeatability (precision) is the ability to tell the same story over and over again or:
 - The error between a number of successive attempts to move the machine to the same position
 - Repeatability is often considered to be the most important parameter of a computer controlled machine (or sensor)
 - Often the intent is to map the errors and then compensate for them
- Minimize static friction, backlash, and thermal variants to get better repeatability



Resolution

- Resolution is how detailed your story is
 - Resolution is the larger of the smallest programmable step or:
 - The smallest mechanical step the machine can make during point to point motion
 - Resolution gives a lower bound on the repeatability
- Minimize static friction and use higher resolution feedback to get better resolution

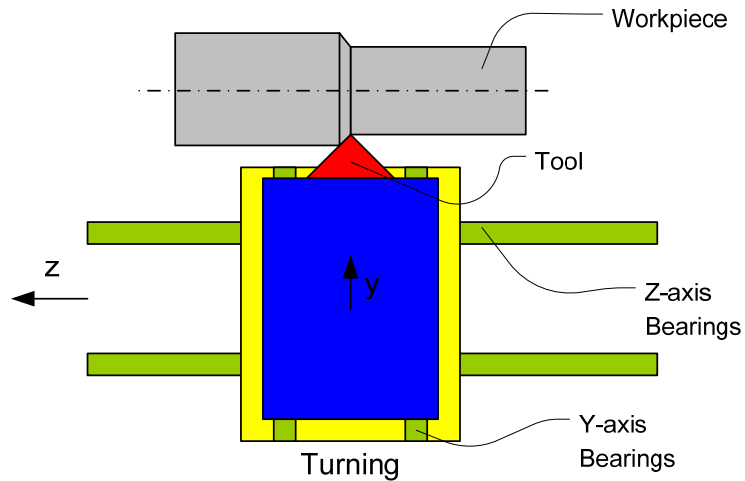


What is the importance of errors?

- The trajectory of the tool relative to the work piece directly determines the dimensions of all features machined
- No machine tool monitors the tool trajectory directly. Instead, the tool trajectory is monitored indirectly by monitoring the position of the moving components (axes)
- Any deviation of any moving component will be imprinted onto the work piece, thereby reducing the part accuracy



Imprinting Form Errors



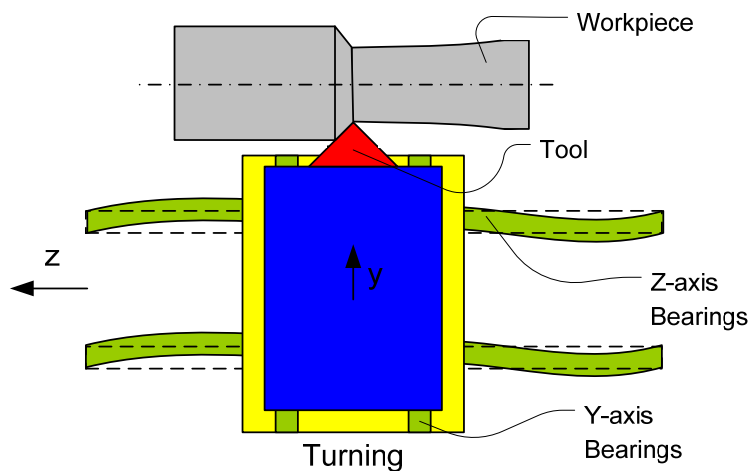
Case 1: Perfectly straight z-axis bearings create a perfectly cylindrical part



ME EN 7960 – Precision Machine Design – Fundamentals of Errors

2-7

Imprinting Form Errors (contd.)



Case 2: The straightness errors of the z-axis bearings are transferred onto the part



ME EN 7960 – Precision Machine Design – Fundamentals of Errors

2-8

Performance Limiting Criteria

- Accuracy and Repeatability are limited by:
 - Geometric errors of all components
 - Kinematic errors
 - Load induced errors
 - Thermal errors
 - Dynamic errors
 - Calibration errors
 - Computational errors
- Resolution is limited by
 - Quality of sensors
 - Quality of control system
 - Friction (stick and slip effect)
 - Backlash



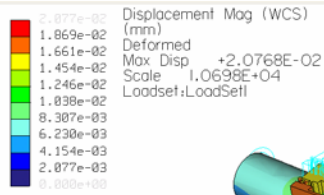
Geometric Errors

- Errors in the form of individual machine components:
 - Component straightness error due to machining errors
 - Component straightness error due to gravity loading
- Surface finish effects
- Quasi-static accuracy of surfaces moving relative to each other (e.g., linear or rotary motion axes):
 - Linear motion axis:
 - Pitch
 - Roll
 - Yaw
 - Straightness (2 components)
 - Linear displacement

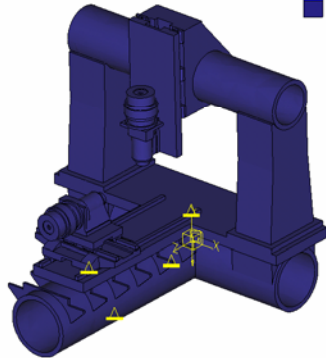
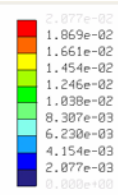


Geometric Error - Gravity

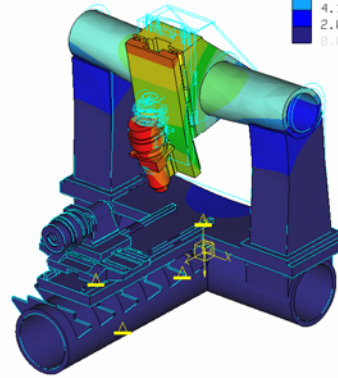
Frame 1 of 8
 Displacement Mag (WCS)
 (mm)
 Deformed
 Max Disp +2.0768E-02
 Scale 1.0698E+04
 Loadset:LoadSet1



Displacement Mag (WCS)
 (mm)
 Deformed
 Max Disp +2.0768E-02
 Scale 1.0698E+04
 Loadset:LoadSet1



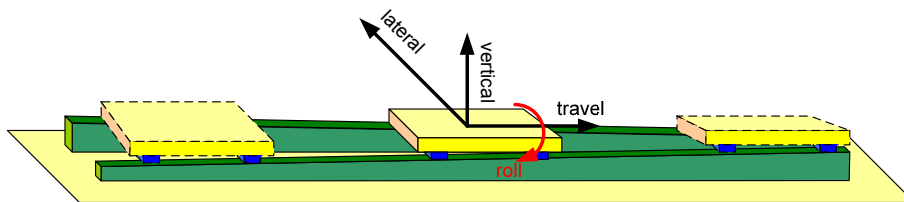
"Window1" - Gravity - Gravity



"Window2" - Gravity - Gravity



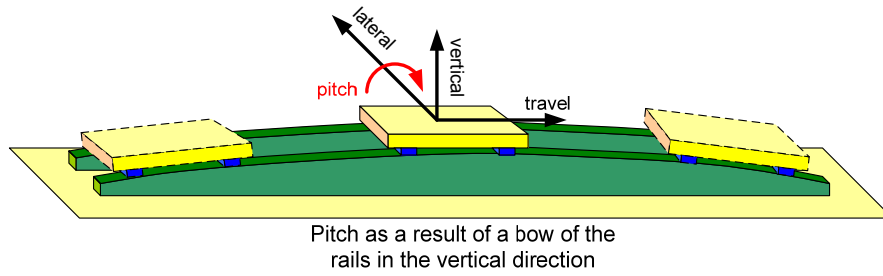
Geometric Errors - Roll



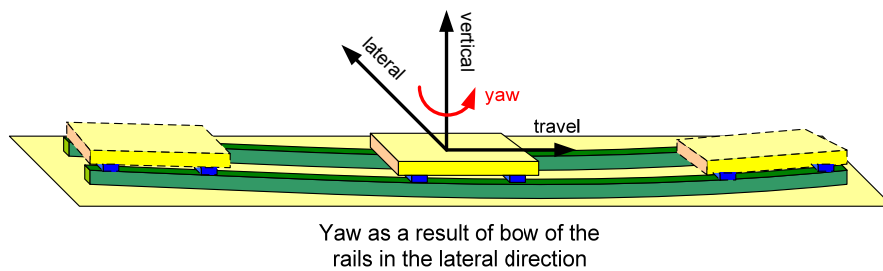
Roll as a result of different slopes of the rails



Geometric Errors - Pitch



Geometric Errors - Yaw

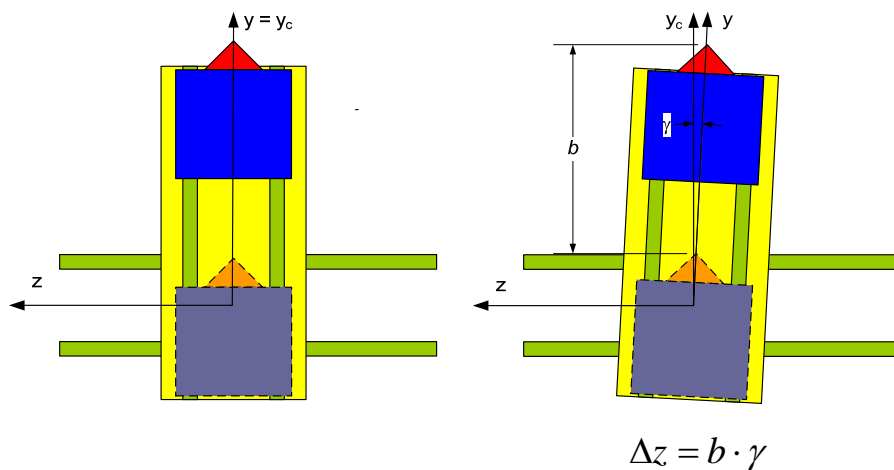


Other Errors

- Kinematic Errors
 - Errors in an axis's trajectory that are caused by misaligned or improperly sized components
 - Orthogonality between axes
 - Parallelism between axes
 - Error motions in a closed kinematic chain
- External load induced errors:
 - Errors due to deformation of components:
 - Gravity load induced errors
 - Cutting/probing force induced errors
 - Axis acceleration load induced errors

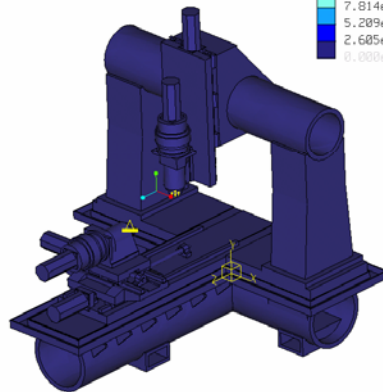
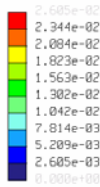


Kinematic Error - Orthogonality Between Axes

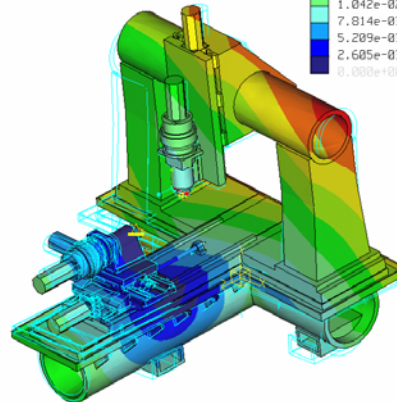
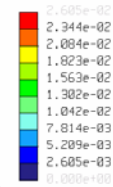


Geometric Error – Load Induced

Frame 1 of 4
 Displacement Mag (WCS)
 (mm)
 Deformed
 Max Disp +2.6046E-02
 Scale 1.0365E+04
 Loadset:LoadSet1



Displacement Mag (WCS)
 (mm)
 Deformed
 Max Disp +2.6046E-02
 Scale 1.0365E+04
 Loadset:LoadSet1



"Window2" - DisplacementAnalysis - DisplacementAnalysis "Window3" - DisplacementAnalysis - DisplacementAnalysis



Thermal Errors

- Mean temperature other than 68 °F (20 °C)
 - Gradients in environment's temperature
 - Errors caused by thermal expansion of elements:
 - External heat sources:
 - Mean temperature of the room
 - Sun shining through the window onto the machine
 - Nearby machine's hot air vent
 - Overhead lights
 - Operator's body heat
 - Internal heat sources:
 - Motors
 - Bearings
 - Machining process
 - Pumps
 - Expansion of compressed fluids
 - Coolant

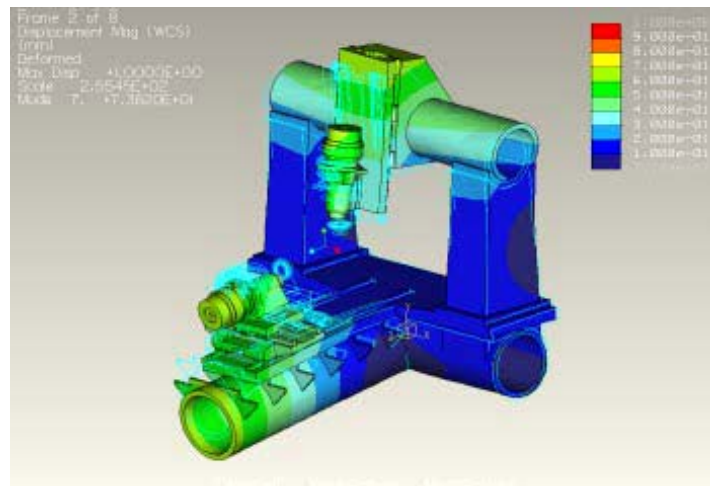


Dynamic Errors

- Errors caused by vibration or control processes:
 - Vibration:
 - External environment (usually through the ground)
 - Cutting process
 - Rotating masses
 - Control system:
 - Algorithm type (e.g., PID, adaptive, etc.)
 - Stick-slip friction
 - Varying mass
 - Varying stiffness
 - Switching amplifiers
 - Servo loop frequency excites a natural mode of the machine



Dynamic Errors



Calibration Errors

- Errors associated with sensors:
 - Intrinsic accuracy
 - Interpolation
 - Mounting errors:
 - Position
 - Mounting stress
 - Calibration (error associated with the mastering process)



Additional Errors

- Computational errors:
 - Error introduced in the analysis algorithms
 - Rounding off errors due to hardware
- Additional sources of error (often very difficult to model):
 - Humidity
 - Loose joints
 - Dirt
- Variations in supply systems:
 - Electricity
 - Fluid pressure
 - Operator inattention
 - Fluid supply cleanliness
- Operators (“52-2” factor)



Error Assessment and Budgeting

- Given all the different types of errors that can affect all different components:
 - Keeping track of all the errors is such a daunting task:
 - Most engineers don't bother and use "experience" to guide the design
 - It is left up to manufacturing and service to work the bugs out
 - This seems to be a major source of reliability and performance problems
- The solution to a successful project is a good budget:
 - A project requires a good **financial budget** to make it feasible
 - A project requires a good **time budget** to make it feasible
 - A project requires a good **error budget** to make it feasible
- In order to make a good error budget for the system, a good mathematical model is needed



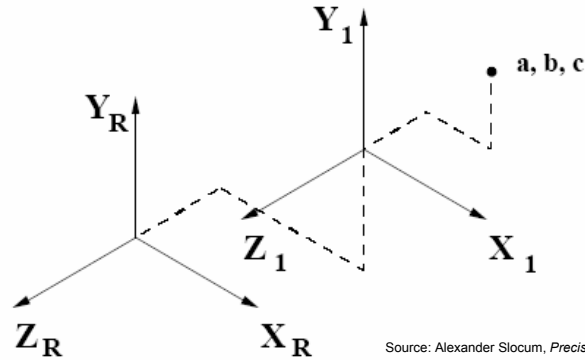
Homogenous Transformation Matrices

- Allows the designer to consider one part of the machine at a time, and then link them all together like beads on a string
- Based on rigid body model of a linear series (open chain) of coordinate frames
- Takes into account linear and angular offsets between coordinate frames
- Transforms XYZ coordinates of one frame into XYZ coordinates of another frame
- Used to transform locally referenced errors into errors referenced with respect to the tool point and the work piece



Homogenous Transformation Matrices (contd.)

- Coordinate frames are placed at bearings, joints, and areas where other parameters are lumped
- Closed chains (e.g. a five point bearing mount) need to be modeled with the generation of constraint equations



Source: Alexander Slocum, *Precision Machine Design*



Structure of HTM's

Orientation of Y_n with respect to adjacent coordinate frame

Orientation of X_n with respect to adjacent coordinate frame

Orientation of Z_n with respect to adjacent coordinate frame

$$R_{T_n} = \begin{bmatrix} O_{ix} & O_{iy} & O_{iz} & P_x \\ O_{jx} & O_{jy} & O_{jz} & P_y \\ O_{kx} & O_{ky} & O_{kz} & P_z \\ 0 & 0 & 0 & 1 \end{bmatrix}$$

Translation of $X_n Y_n Z_n$ with respect to adjacent coordinate frame



Structure of HTM's (contd.)

- The first three columns are the direction cosines (unit vectors i, j, k).
 - They represent the orientation of the $X_n, Y_n,$ and Z_n axes with respect to an adjacent coordinate frame
- The last column is the position of the rigid body's coordinate system's origin with respect to the reference frame
- The pre-superscript represents the reference frame in which you want the result to be represented
- The post-subscript represents the reference frame from which you are transferring:



Structure of HTM's (contd.)

- The equivalent coordinates of a point in a reference frame n , in a reference frame R are:

Equivalent coordinates of a point in reference frame n , in reference frame R

Equivalent coordinates of a point in reference frame n , in reference frame R

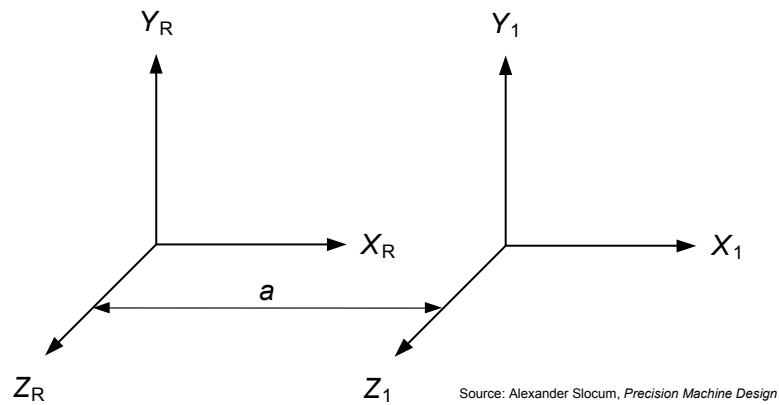
$$\begin{bmatrix} X_R \\ Y_R \\ Z_R \\ 1 \end{bmatrix} = R_{T_n} \begin{bmatrix} X_n \\ Y_n \\ Z_n \\ 1 \end{bmatrix}$$

HTM that describes the transformation of reference frame n into reference frame R



Example: Translation in X

- The coordinate system $X_1Y_1Z_1$ is shifted in X by an amount a :



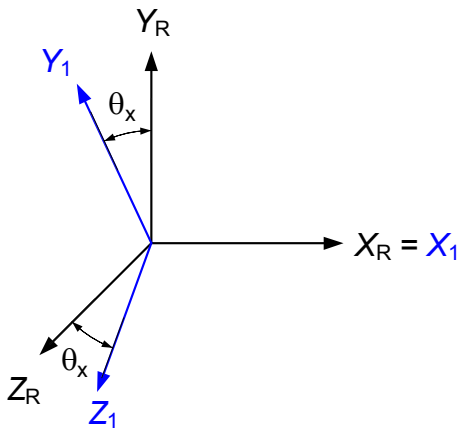
HTM of Translation in x

$$XYZ_{T_{X_1Y_1Z_1}} = \begin{bmatrix} 1 & 0 & 0 & a \\ 0 & 1 & 0 & 0 \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}$$



Example: Rotation about X-Axis

- the $X_1Y_1Z_1$ coordinate system is rotated by an amount θ_x about the X axis:



HTM of Rotation about X

$$XYZ_{T_{X_1Y_1Z_1}} = \begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & \cos \theta_x & -\sin \theta_x & 0 \\ 0 & \sin \theta_x & \cos \theta_x & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}$$



Sequential Systems

- Transformation from the Nth axis to the reference system will be the sequential product of all the HTMs:

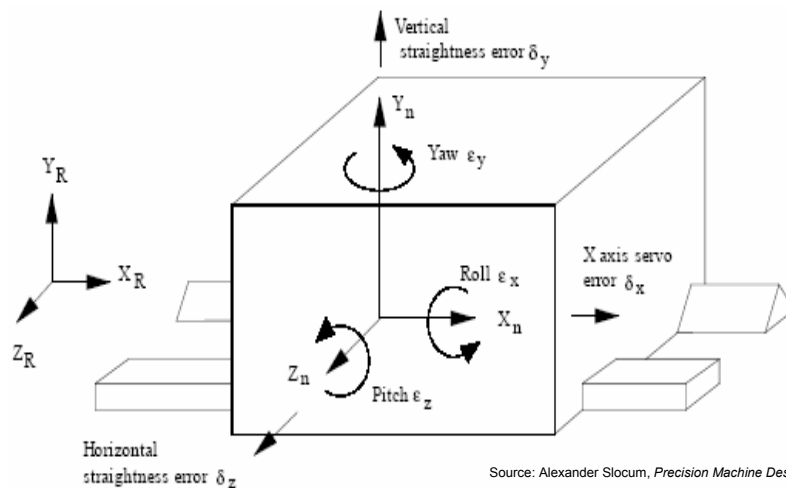
$$R_{T_N} = \prod_{m=1}^N {}^{m-1}T_m = {}^0T_1 {}^1T_2 {}^2T_3 \dots$$

- The error between the tool and the work piece is obtained from:

$$\begin{bmatrix} \Delta x \\ \Delta y \\ \Delta z \end{bmatrix} = \begin{bmatrix} P_x \\ P_y \\ P_z \end{bmatrix}_{work,R} - \begin{bmatrix} P_x \\ P_y \\ P_z \end{bmatrix}_{tool,R}$$



Rigid body models of machine components

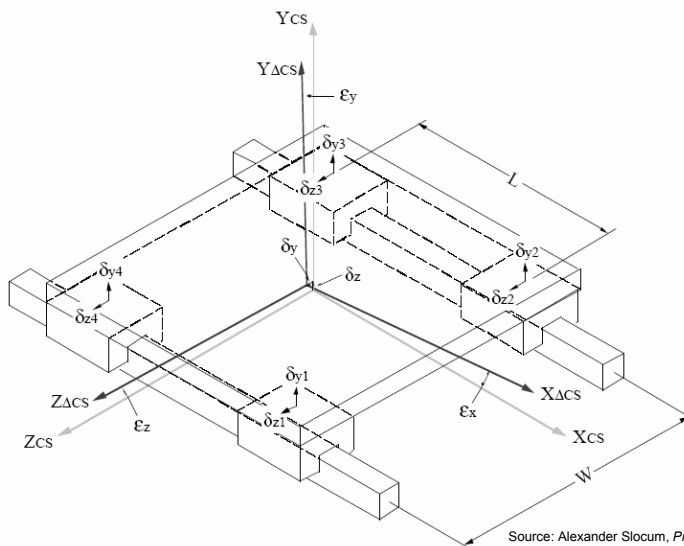


HTM of Linear Motion Carriage

$$R_{T_{nerr}} = \begin{bmatrix} 1 & -\varepsilon_z & \varepsilon_y & a + \delta_x \\ \varepsilon_z & 1 & -\varepsilon_x & b + \delta_y \\ -\varepsilon_y & \varepsilon_x & 1 & c + \delta_z \\ 0 & 0 & 0 & 1 \end{bmatrix}$$



Estimating Position Errors from Catalogs

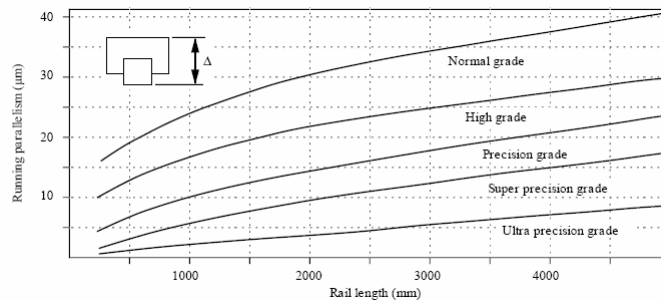


HTM of Modular Bearing System

$$\begin{bmatrix} 1 & -\varepsilon_z & \varepsilon_y & a + \delta_x \\ \varepsilon_z & 1 & -\varepsilon_x & b + \delta_y \\ -\varepsilon_y & \varepsilon_x & 1 & c + \delta_z \\ 0 & 0 & 0 & 1 \end{bmatrix}$$



HTM of Modular Bearing System (contd.)



Source: Alexander Slocum, *Precision Machine Design*

The translational errors are based on the average of the straightness errors experienced by the bearing block:

$$\delta_{axial} = \delta_{servo}$$

$$\delta_{lateral} = \frac{\delta_{lateral1} + \delta_{lateral2} + \delta_{lateral3} + \delta_{lateral4}}{4}$$

$$\delta_{vertical} = \frac{\delta_{vertical1} + \delta_{vertical2} + \delta_{vertical3} + \delta_{vertical4}}{4}$$



HTM of Modular Bearing System (contd.)

- The angular errors are based on the differences in the average straightness errors experienced by pairs of bearing blocks acting across the carriage:

$$\mathcal{E}_{axial} = \frac{\frac{\delta_{vertical2} + \delta_{vertical3}}{2} - \frac{\delta_{vertical1} + \delta_{vertical4}}{2}}{W}$$
$$\mathcal{E}_{lateral} = \frac{\frac{\delta_{vertical1} + \delta_{vertical2}}{2} - \frac{\delta_{vertical3} + \delta_{vertical4}}{2}}{L}$$
$$\mathcal{E}_{vertical} = \frac{\frac{\delta_{lateral3} + \delta_{lateral4}}{2} - \frac{\delta_{lateral1} + \delta_{lateral2}}{2}}{L}$$

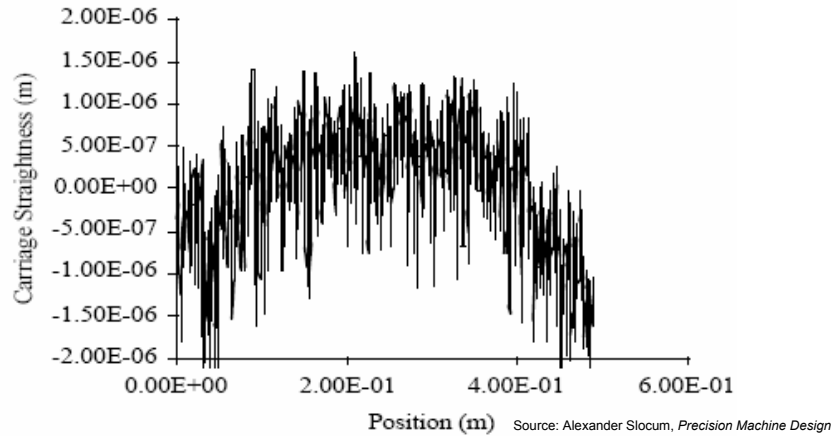


HTM of Modular Bearing System (contd.)

- Note we assumed for the straightness we assumed all the errors are acting in the same direction
- Here we assume one set of errors acts up, and the other acts down
- This is very conservative (makes up for other effects we might miss)



Linear Error Plot

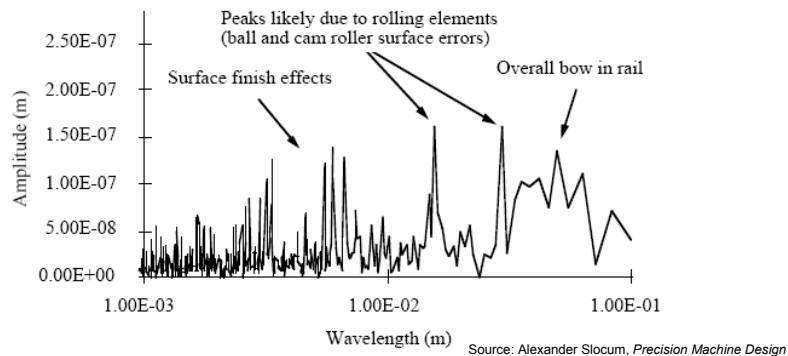


Straightness error of a kinematically (statically determined) supported linear axis (5 rolling element bearings on a vee and flat)



Fourier Transformation of Linear Error Plot

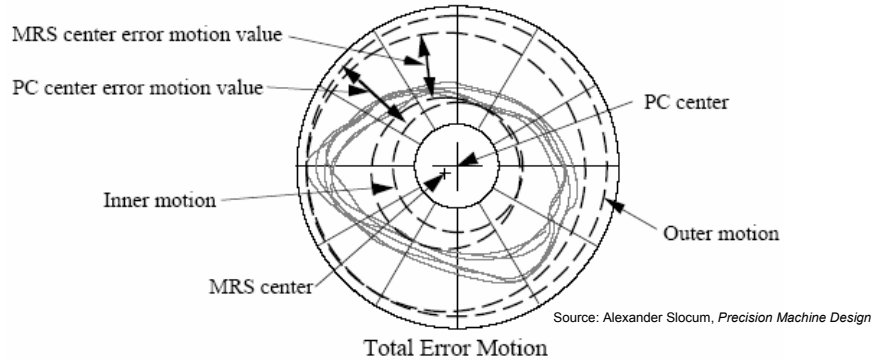
- The Fourier Transformations when plotted as the error amplitude as a function of wavelength is an invaluable diagnostic tool



- The wavelength of rolling elements is $2\pi D$



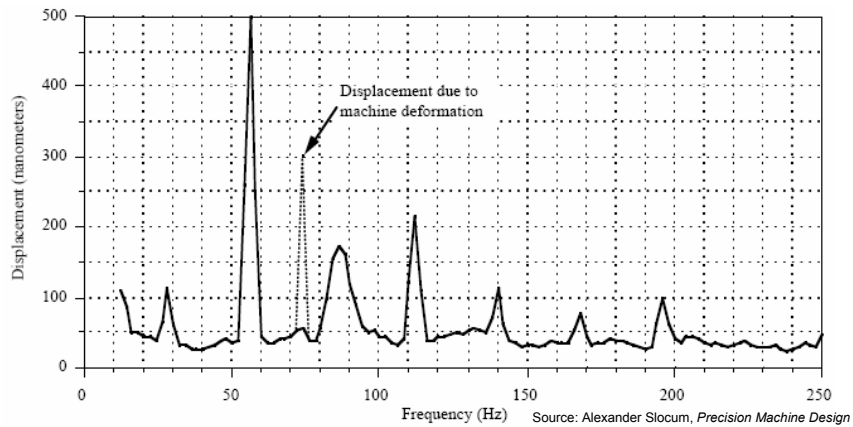
Rotary Error Plot



- The average error motion is indicative of the form error that will be imparted to the part when held in a lathe spindle
- The asynchronous error motion is indicative of the surface finish that will be obtained



Fourier Transformation of Radial Error



- Spindle speed: 1680 rpm (28 Hz)
- Bearing ID: 75 mm
- Bearing OD: 105 mm
- Number of balls: 20
- Ball diameter: 10 mm
- Contact angle: 15



Combinational Ruler for 3 Common Types

- **Random** - under apparently equal conditions at a given position, errors that do not always have the same value, and can only be expressed statistically
- **Systematic** - which always have the same value and sign at a given position and under given circumstances
 - Generally can be correlated with position along an axis and can be corrected
 - If the relative accompanying random error is small enough
- **Hysteresis** - a systematic error which in this instance is separated out for convenience
 - Usually repeatable, sign depends on the direction of approach, and magnitude partly dependent on the travel
 - May be compensated for if the direction of approach is known and an adequate pre-travel is made

